

Dynamics of Delayed Piecewise Linear Mathieu Equation

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Abstract. We study the dynamics of a piecewise linear (PWL) oscillator subjected to parametric excitation, time delayed feedback and cubic nonlinear interaction force. The governing equation is non-smooth, essentially nonlinear and infinite-dimensional. In the absence of cubic nonlinearity, the solutions are scalable, i.e., if $\phi(t)$ is a solution, then $\alpha\phi(t)$ ($\alpha \neq 0$) is also a solution. We render the dynamical system finite dimensional by using Galerkin approximation and evaluate the Lyapunov-like exponent to explore the regions of stability. The method of averaging (MAV) is invoked to derive slow-flow equations to explore the stability of periodic solutions and bifurcations thereof.

Introduction

PWL oscillators are isochronous in spite of their essential nonlinearity. A cracked beam [1] exhibits PWL behaviour wherein the effective stiffness is higher during crack closure phase in comparison to that of crack opening. Systems which have intermittent contact and/or backlash [2] are effectively modeled as PWL systems. As such, analytical study of their dynamical behavior is of interest and importance. To this end, we consider a PWL Mathieu equation [2] [3] with cubic nonlinearity and time delay in the following non-dimensional form

$$\ddot{u} + \left\{ \kappa(u) + \varepsilon P \sin(\Omega t) \right\} u + \varepsilon C u^3 = \varepsilon D u(t-1). \quad (1)$$

Where time delay has been rescaled to unity, P, C , and D scaled by ε ($0 \leq \varepsilon \ll 1$) are the amplitude of the parametric excitation, strength of cubic nonlinearity, and delayed feedback respectively, Ω is the frequency of the parametric excitation, $\kappa(u) = k_1^2$ for $u > 0$ and $\kappa(u) = k_2^2$ for $u \leq 0$. The time period and natural frequency of the unperturbed autonomous oscillator ($\varepsilon = 0$) is $T = \pi(1/k_1 + 1/k_2)$, $\omega_{pwl} = 2\pi/T$ respectively. The excitation frequency is considered close to a resonance manifold such that $\Omega = m\omega_{pwl} + \varepsilon\sigma$, where $m \in \mathbb{Z}^+$ and $\sigma = O(1)$ is the frequency detuning parameter.

Results and discussion

We begin with the linear system ($C = 0$) and use Galerkin approximation to render the infinite-dimensional dynamical system a finite-dimensional one. Owing to the scalability of the system, we evaluate the Lyapunov-like exponents [3] to explore the stable and unstable regions in the $\sigma - P$ plane (Fig. 1(a)). MAV is invoked by considering PWL basis functions [4] for the unperturbed autonomous system and derive the slow-flow equations. The fixed points of the slow-flow equations correspond to the steady state solutions of Eq. (1) and forms the boundary in Fig. 1(a) (green curve). In case of a nonlinear system ($C \neq 0$), there are multiple steady state solutions and the bifurcation plot is shown in Fig. 1(b). The steady state solutions undergo saddle-node bifurcation at $\sigma = \sigma_1$, supercritical pitchfork bifurcation at $\sigma = \sigma_2$, subcritical pitchfork bifurcation at $\sigma = \sigma_3$ and saddle-node bifurcation at $\sigma = \sigma_4$. From the MAV, we observe that there exists no trivial steady-state solutions before $\sigma = \sigma_1$ and after $\sigma = \sigma_4$ in Fig. 1(b). Fig. 1(c) shows the three-dimensional bifurcation plot in $\sigma - k_2 - A_a^*$ space. By decreasing the value k_2 to 1.5174, we eliminate subcritical pitchfork bifurcation. Below $k_2 = 1.5174$, we have only two saddle-node bifurcation of the trivial solution $A_a^* = 0$.

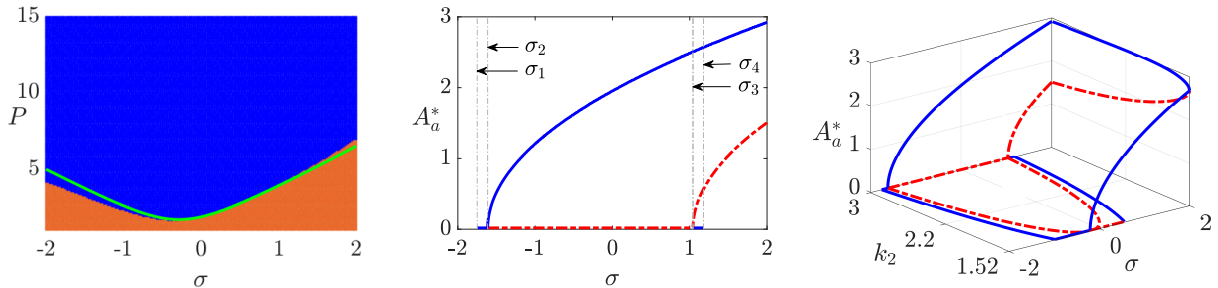


Figure 1: (a) Stability chart, orange: stable, blue: unstable and green: steady state solution of slow-flow equation ($C = 0$). (b) Bifurcation diagram for $k_2 = 3$. (c) Bifurcation diagram in $\sigma - k_2 - A_a^*$ space, (dotted lines: unstable, and solid lines: stable solutions) corresponding to $k_1 = 1$, $\varepsilon = 0.1$, $P = 4$, $C = D = 1$, $m = 1$.

References

- [1] Chati M., Rand R., Mukherjee S. (1997) Modal analysis of a cracked beam. *Jl. of Sound and Vibration* **207**(2):249-270.
- [2] Theodossiades S., Natsiavas S. (2000) Nonlinear dynamics of gear pair systems with periodic stiffness and backlash. *Jl. of Sound and Vibration* **229**(2):287-310.
- [3] Marathe A., Chatterjee A. (2006) Asymmetric Mathieu equations. *Proc. of the Royal Soc. A* **462**(2070):1643-1659.
- [4] Jayaprakash K. R., Tandel V., Starosvetsky Y. (2022) Dynamics of excited piecewise linear oscillators. *Nonlin. Dyn.* (in revision).